## 6. PUMPS AND PUMPING SYSTEM

Types, Performance evaluation, Efficient system operation, Flow control strategies, Energy conservation in boiler feed water pump, pumping systems for municipal drinking water, sewage water pumps, agriculture pump sets, energy conservation opportunities.

## 6.1 Pump Types

Pumps come in a variety of sizes for a wide range of applications. They can be classified according to their basic operating principle as dynamic or displacement pumps. Dynamic pumps can be sub-classified as centrifugal and special effect pumps. Displacement pumps can be sub-classified as rotary or reciprocating pumps.

In principle, any liquid can be handled by any of the pump designs. Where different pump designs could be used, the centrifugal pump is generally the most economical followed by rotary and reciprocating pumps. Although, positive displacement pumps are generally more efficient than centrifugal pumps, the benefit of higher efficiency tends to be offset by increased maintenance costs.

Since, worldwide, centrifugal pumps account for the majority of electricity used by pumps, the focus of this chapter is on centrifugal pump.

## **Centrifugal Pumps**

A centrifugal pump (Figure 6.1) is of a very simple design. The two main parts of the pump are the impeller and the diffuser. Impeller, which is the only moving part, is attached to a shaft and driven by a motor. Impellers are generally made of bronze, polycarbonate, cast iron, stainless steel as well as other materials. The diffuser (also called as volute) houses the impeller and captures and directs the water off the impeller.

Water enters the center (eye) of the impeller and exits the impeller with the help of centrifugal force. As water leaves the eye of the impeller a low-pressure area is created, causing more water to flow into the eye. Atmospheric pressure and centrifugal force cause this to happen. Velocity is developed as the water flows through the impeller spinning at high speed. The water velocity is collected by the diffuser and converted to pressure by specially designed passageways that direct the flow to the discharge of the pump, or to the next impeller should the pump have a multi-stage configuration.

The pressure (head) that a pump will develop is in direct relationship to the impeller diameter, the number of impellers, the

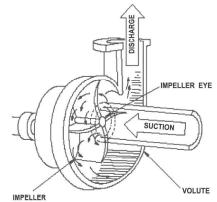


Figure 6.1 Centrifugal Pump

size of impeller eye, and shaft speed. Capacity is determined by the exit width of the impeller. The head and capacity are the main factors, which affect the horsepower size of the motor to be used. The more the quantity of water to be pumped, the more energy is required.

A centrifugal pump is not positive acting, it will not pump the same volume always. The greater the depth of the water, the lesser is the flow from the pump. Also, when it pumps against increasing pressure, the less it will pump. For these reasons it is important to select a centrifugal pump that is designed to do a particular job.

Since the pump is a dynamic device, it is convenient to consider the pressure in terms of head i.e. meters of liquid column. The pump generates the same head of liquid whatever the density of the liquid being pumped. The actual contours of the hydraulic passages of the impeller and the casing are extremely important, in order to attain the highest efficiency possible. The standard convention for centrifugal pump is to draw the pump performance curves showing Flow on the horizontal axis and Head generated on the vertical axis. Efficiency, Power & NPSH Required (described later), are also all conventionally shown on the vertical axis, plotted against Flow, as illustrated in Figure 6.2.

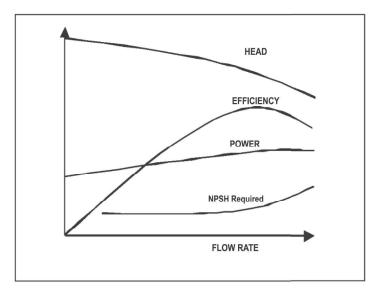


Figure 6.2 Pump Performance

Given the significant amount of electricity attributed to pumping systems, even small improvements in pumping efficiency could yield very significant savings of electricity. The pump is among the most inefficient of the components that comprise a pumping system, including the motor, transmission drive, piping and valves.

## Hydraulic Power, Pump Shaft Power and Motor Input Power

 $\textbf{Hydraulic Power P}_{h} = Q \; (m^3/s) \; x \; \text{Total Differential head, } \\ h_{d} - h_{s} \; (m) \; x \; \rho \; (kg/m^3) \; x \; g \; (m/s^2) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; (m/s) \; / \; 1000 \; (kg/m^3) \; x \; ($ 

Where  $h_d$  - discharge head,  $h_s$  - suction head,  $\rho$  - density of the liquid, g - acceleration due to gravity

 $\textbf{Pump Shaft Power P}_{s} = \text{Hydraulic power, P}_{h} \text{ / Pump Efficiency, } \eta_{\text{Pump}}$ 

**Motor Input Power** = Pump shaft power  $P_s$  / Motor Efficiency,  $\eta_{Motor}$